# Computation of $\mathbf{N} \gamma$ for a Smooth Footing - Hill Mechanism 

D. M. Dewaikar*, B. G. Mohapatro**, V. A. Sawant*** and H. S. Chore****

## Introduction

T
he bearing capacity of shallow foundations is one of the most studied areas in the field of soil-structure interaction problems. The foundation bases may be rough or smooth depending on the assumptions made in the analysis. Different failure mechanisms have been proposed for the analysis of bearing capacity of shallow foundations, the most common amongst them being the mechanisms postulated by Prandtl (1920), Terzaghi (1943) and that by Hill (1950). While the mechanism proposed by Terzaghi provided the first comprehensive bearing capacity analysis for a rough strip footing, the mechanisms postulated by Prandtl (1920) dealt with the strip footing with a smooth base. Hill (1950) proposed a different mechanism than that by Prandtl (1920) for the punch - indentation problem which is applicable for smooth footings. This mechanism gives a better estimate of bearing capacity factors in respect of cohesive - frictional ( $\mathrm{c}-\phi$ ) ponderable (loaded with self weight) soils. Based on the mechanism postulated by Hill (1950), several theories have been proposed by researchers (Chen 1975; Michalowski 1997) to determine the bearing capacity factor- $\mathrm{N}_{\gamma}$ in respect of a ponderable soil.

In this paper, computations of bearing capacity factor $\mathrm{N}_{\gamma}$ based on the failure mechanism postulated by Hill for smooth footings, using limit equilibrium approach coupled with Kötter's equation are presented and the results are compared with available solutions.

## Brief Review of Hill Mechanism

Mechanism suggested by Hill for evaluating bearing capacity factor for the strip footing with a smooth base is the modification in the classical Prandtl's mechanism. Hill modified Prandtl's Mechanism as shown in Figure 1(a). His

[^0]mechanism is symmetric about the axis of footing. It consists of: (i) a triangular wedge ABC with base angles, $\psi=\pi / 4+\phi / 2$, and the interface between the wedge and the footing bottom is considered to be smooth, thus, allowing the horizontal movement of the wedge; (ii) a log spiral shear zone BCD of central angle $\pi / 2$; and (iii) a passive Rankine triangular wedge BDE with base angles $\pi / 4-\phi / 2$.

## Formulation of the Problem

The analysis is primarily based on the computation of vertical ( $\mathrm{R}_{\mathrm{V}}$ ) and horizontal $\left(\mathrm{R}_{H}\right)$ components of reaction ( R ) that acts on the curved part CD of the failure surface [Figure 1 (b)]. For this purpose Kötter's equation (1903) is used.

## Kötter's Equation

For a cohesion-less soil medium (friction angle $\phi$, unit weight $\gamma$ ) in passive state of equilibrium, Kötter's equation gives the distribution of soil pressure p along the arc of the failure surface in the following form:

$$
\begin{equation*}
\frac{d p}{d s}+2 p \tan \phi \frac{d \alpha}{d s}-\gamma \sin (\alpha+\phi)=0 \tag{1}
\end{equation*}
$$

in which, $d p$ is differential reactive pressure on the elemental length $d s$ of the failure surface, and $\alpha$ is angle made by the tangent to failure surface at the point of interest with the horizontal.

The applicability of Kötter's equation to the analysis of the limit equilibrium problems has been demonstrated for the problems of a retaining wall using Coulomb's mechanism for the case of ponderable cohesion-less soil by Dewaikar and Halkude (2002).

## Outline of the Proposed Analysis

For the analysis, the right half of the symmetry with width equal to $2 B$ [Figure 1(a)] is considered. The forces that act on the failure wedge CDFB are vertical $\left(R_{V}\right)$ and horizontal $\left(R_{H}\right)$ components of reaction R on curved part CD of the failure surface, $P_{\gamma}$ (passive thrust exerted on the Rankine's Wall) and $W$ (self weight of the wedge CDFB). The analysis is primarily based on the computation of $R_{V}$ and $R_{H}$ using Kötter's equation. After computing these reactions, free body diagram of wedge CDFB, is considered. In this free body diagram, $P_{p \gamma}$ is the only unknown force. Now, if the pole of the log spiral CD is correctly located, the calculated passive thrust $P_{p \gamma}$ from the force equilibrium conditions will be exactly the same, otherwise, they will be different. If they are different, trial location of the pole along the line BD is changed and for this new location of the pole, $R_{H}, R_{V}, W$ and $P_{\gamma}$ are again computed. Iterations are thus
continued till the passive thrust $P_{p \gamma}$ obtained from both force equilibrium condition is the same to a specified decimal accuracy. After satisfying this condition, a unique value of $P_{p \gamma}$ is obtained, from which $N_{\gamma}$ is computed.

The point of application of $P_{p \gamma}$ is computed using moment equilibrium condition in which moments of all the forces and reactions about the pole of log spiral are considered. It is convenient since the resultant reaction (R) always acts along the radial line of the log spiral and makes no contribution to the moment. Thus, in the proposed analysis, force equilibrium conditions are used to compute $P_{p \gamma}$ and moment equilibrium condition is used to compute its point of application.


Fig. 1 (a) Failure Mechanism- Hill's Analysis;
(b) Free- body Diagram of Wedge CDFB

## Integration of Kötter's Equation over the Part DE of the Failure Surface

This part of the failure surface being straight, $d \alpha / d s=0$ and Kötter's equation reduces to the following form.
$\frac{d p}{d s}=\gamma \sin (\alpha+\phi)$
Integration of the above equation gives the soil reaction pressure at a point on the plane failure surface $D E$ and the value of $p$ at point $D$ is calculated as

$$
\begin{equation*}
p=\gamma \sin (\pi / 4+\phi / 2) D E \tag{3}
\end{equation*}
$$

The distance DE depends upon the location of pole of the log spiral as shown in Figure 2. Referring to Figures 1 and 2, DE is calculated from geometry of the failure wedge. With this substitution Equation (3) becomes
$p=\gamma \sin (\pi / 4+\phi / 2) K r_{0} e^{\theta_{m} \tan \phi}$
in which, $r_{0}, \theta_{m}$, and $\theta_{v}$ are as shown in Figure 2 and K is given by the following expression:
$K=\left(1+\frac{\sin \theta_{v}}{e^{\theta_{m} \tan \phi}}\right)$


Fig. 2 Geometrical Relationships for Pole below Footing Base


Fig. 3 Geometrical Relations for the Determination of Point of Application of Passive Thrust

For this purpose, wedge CDFB as shown in Figure 1 (b) is referred. The magnitude of passive Rankine thrust, $P_{\gamma}$ is given as:

$$
\begin{equation*}
P_{\gamma}=\frac{1}{2} \gamma(D F)^{2}\left(\frac{1+\sin \phi}{1-\sin \phi}\right) \tag{6}
\end{equation*}
$$

Among the above forces, the known forces are $P_{\gamma}$ and $W$ and the forces $R_{H} \quad$ and $\quad R_{V}$ are calculated using Kötter's equation for a curved failure surface (Figure 2).

Integration of the Kötter's equation gives the pressure distribution on the curved failure surface (Mohapatro 2001) and is given as

$$
\begin{align*}
& p=\left\{\gamma r_{0} K \cos \phi \sin (\pi / 4+\phi / 2) e^{\left(3 \theta_{m}-2 \theta\right) \tan \phi}\right\}+ \\
& \frac{\gamma r_{0} \sec \phi}{\left(1+9 \tan ^{2} \phi\right)}\left[\begin{array}{l}
e^{\theta \tan \phi}\left\{\begin{array}{l}
3 \tan \phi \sin \left(\theta-\theta_{L}+\phi\right) \\
-\cos \left(\theta-\theta_{L}+\phi\right)
\end{array}\right\}- \\
e^{\left(3 \theta_{m}-2 \theta\right) \tan \phi}\left\{\begin{array}{l}
3 \tan \phi \sin \left(\theta_{m}-\theta_{L}+\phi\right) \\
-\cos \left(\theta_{m}-\theta_{L}+\phi\right)
\end{array}\right\}
\end{array}\right\} \tag{7}
\end{align*}
$$

where $\theta_{L}=\pi / 4+\phi / 2+\theta_{v}$ and $\theta$ is as shown in Figure 2. In deriving the above expression, reactive pressure at point $D$ as given by Equation (4) is used as a boundary condition.

## Components of Resultant Reaction on the Failure Surface

The resultant reaction, $R$ on the failure surface is given as

$$
\begin{equation*}
R=\int p d s \tag{8}
\end{equation*}
$$

The vertical component, $R_{V}$ of the reaction is obtained as (Figure 2),
$R_{V}=\int_{0}^{\theta_{m}} p r_{0} e^{\theta \tan \phi} \cos \left(\theta-\theta_{L}+\phi\right) \sec \phi d \theta$
After substituting the value of $p$ from Equation (7), the value $R_{V}$ is obtained in the following form:

$$
\begin{equation*}
R_{V}=I_{1}+I_{2}+I_{3} \tag{10}
\end{equation*}
$$

Similarly, the horizontal component, $R_{H}$ of the resultant reaction is given as (Figure 2)
$R_{H}=\int_{0}^{\theta_{m}} p \sin \left(\theta-\theta_{L}+\phi\right) r_{0} e^{\theta \tan \phi} \sec \phi d \theta$
After substituting the value of $p$ from Equation (7) and performing integration, $R_{H}$ is obtained as

$$
\begin{equation*}
R_{H}=I_{4}+I_{5}+I_{6} \tag{12}
\end{equation*}
$$

The details of parameters $I_{1}, I_{2}, I_{3}, I_{4}, I_{5}$ and $I_{6}$ that appear in Equations (11) and (12) are reported in Appendix.

## Self -weight of the Wedge CDFB

This is obtained by calculating weights, $W_{1}$ of part OCD, $W_{2}$ of part OCB and $W_{3}$, of part BDF (Figure 3). The required weight $W$ (of part CDFB) is given as
$W=W_{1}+W_{2}+W_{3}$
In which,
$W_{1}=\frac{1}{4} \frac{\gamma r_{0}^{2}}{\tan \phi}\left[e^{2 \theta_{m} \tan \phi}-1\right] ; \quad W_{2}=\frac{1}{4} \gamma r_{0}^{2} \sin 2 \theta_{v} ;$
$W_{3}=\frac{1}{4} \gamma r_{0}^{2} K^{2} e^{2 \theta_{m} \tan \phi} \cos \phi$

## Computation of Passive Thrust $P_{p \gamma}$

The passive thrust is obtained by using the two equations of force equilibrium [Figure 1(b)]. The vertical force equilibrium is

$$
\begin{equation*}
R_{V}-P_{p \gamma} \cos (\pi / 4-\phi / 2)-W=0 \tag{15}
\end{equation*}
$$

The horizontal force equilibrium is
$P_{p \gamma} \sin (\pi / 4-\phi / 2)-R_{H}-P_{\gamma}=0$
The procedure of obtaining the desired value of $P_{p \gamma}$ has been described in the outline of the proposed analysis. The iterations were carried out till the computed values of $P_{p \gamma}$ from both the equations of force equilibrium matched with each other up to four decimal places.

## Computation of $N_{\gamma}$

For this computation, Figure 4 is referred, which shows the free body diagram of the triangular wedge, $A B C$, subjected to forces $Q_{u l}(k N / m), W_{s}$
(Weight of the triangular soil wedge, ABC ) and vertical component of the passive thrust, $P_{p \gamma}$. The vertical force equilibrium gives:
$Q_{u l}+W_{s}=2 P_{p \gamma} \cos (\pi / 4-\phi / 2)$
After substituting the value of $W_{s}$ in Equation (17) and dividing by $2 B$, it becomes:

$$
\begin{equation*}
\frac{Q_{u l}}{2 B}=q_{u}=\frac{1}{2 B}\left\{2 P_{p \gamma} \cos (\pi / 4-\phi / 2)-\gamma B^{2} \tan (\pi / 4+\phi / 2)\right\} \tag{18}
\end{equation*}
$$

Considering the total footing width equal to 4 B , the ultimate bearing pressure, $\mathrm{q}_{\mathrm{u}}$ for a strip footing on the surface of a cohesion-less soil, with unit weight of the soil $\gamma$ is given by
$q_{u}=2 \gamma B N_{\gamma}$

On comparing Equations (18) and (19), the bearing capacity factor $N_{\gamma}$ is finally obtained as

$$
\begin{equation*}
N_{\gamma}=\frac{1}{2}\left\{\frac{P_{p \gamma}}{\gamma B^{2}} \cos (\pi / 4-\phi / 2)-\frac{1}{2} \tan (\pi / 4+\phi / 2)\right\} \tag{20}
\end{equation*}
$$



Fig. 4 Free -body Diagram Triangular Wedge ABC for Determination of $N_{\gamma}$

## Point of Application of $P_{p \gamma}$

For this purpose, wedge CDFB as shown in Figure 3 is referred. This is obtained by calculating centroids, $\bar{x}_{1}$ of area OCD (weight $\mathrm{W}_{1}$ ), $\bar{x}_{2}$ of area OCB (weight $\mathrm{W}_{2}$ ), and $\bar{X}_{3}$ of area BDF (weight $\mathrm{W}_{3}$ ). Then centroid of the wedge CDFB with reference to $X, Y$ axes is given as

$$
\begin{equation*}
\bar{x}=\frac{W_{1} \bar{X}_{1}+W_{2} \bar{x}_{2}+W_{3} \bar{X}_{3}}{W_{1}+W_{2}+W_{3}} \tag{21}
\end{equation*}
$$

The passive thrust, $P_{p \gamma}$ acts on the line, CB at point G with angle $\phi$, to the normal. Unknown distance BG of point of application of $P_{p \gamma}$ from the pole O , can be obtained from moment equilibrium. Taking moment of all forces about the pole, O , the following equation is obtained after simplification.

$$
\begin{equation*}
P_{p \gamma}(B G \cos \phi-O B \sin \phi)=W \bar{x}+P_{\gamma}\left(\frac{2}{3} B D-O B\right) \sin (\pi / 4-\phi / 2) \tag{22}
\end{equation*}
$$

## Results and Discussion

In Table 1, values of $N_{\gamma}$ as obtained from the proposed analysis are presented for different values of the angle of internal friction (in the range of $20^{\circ}$ $50^{\circ}$ ) along with those reported by other researchers. It is seen that, the $N_{\gamma}$ values obtained by the proposed method are lower than those computed by Sokolovski (1965) based on method of characteristics, in the range of 5 to 13 per cent, for $\phi$ values greater than $20^{\circ}$.

The comparison with $N_{\gamma}$ values reported by Chen (1975) which are based on the upper bound theory of limit analysis show a closer agreement to the values obtained by the proposed method. For $\phi \geq 25^{\circ}$, they are lower than the proposed values in the range of 5 to 11.5 percent.

The $N_{\gamma}$ values obtained by the proposed method are higher than those reported by Bolton and Lau (1993) based on the method of characteristics for $\phi \geq 25^{\circ}$, in the range of 70 to 87 per cent. The proposed $N_{\gamma}$ values are higher to the tune of 25 to 31 per cent than those reported by Michalowski (1997) using upper bound theory of limit analysis. Similarly, $N_{\gamma}$ values as obtained by the proposed method are higher than the values reported by Frydman and Burd (1997), based on finite difference method coupled with associated flow rule, in the range of 34 to 61 per cent.

Thus, from above comparison, it is seen that the proposed $N_{\gamma}$ values show a good agreement with the values reported by Sokolovski (1965) and Chen (1975); and so far as the computations using limit equilibrium method are concerned, the proposed $N_{\gamma}$ values are the only available values in the literature, in the context of Hill's mechanism.

The position of the pole (corresponding to the unique failure surface) is indicated by the values of angle, $\theta_{\mathrm{v}}$ (Figure 3), which is reported in Table 2 (refer column 2). From these values of angles $\left(\theta_{v}\right)$, it is seen that for general
shear failure conditions, which exists beyond $\phi=36^{\circ}$, the pole of the log spiral lies very close to the footing edge.

The location of point of application of the passive thrust $\left(P_{p \gamma}\right)$ is possible using the proposed method only. It is expressed as the ratio of its vertical distance from Point C (Figure 3) and from the footing base. It increases with an increase in $\phi$ values (refer column 3 of Table 2).

TABLE 1: Comparison of Bearing Capacity Factor ( $N_{\gamma}$ ) with other theories (Smooth Footing)
(L. B. = Limit Equilibrium; C = Characteristics; U. B. = Upper Bound; K = Kinematic;
N. M. = Numerical Method)

| $\begin{gathered} \phi \\ \text { (Deg) } \end{gathered}$ | $N_{\gamma}$ |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | Proposed Analysis | Sokolovski (1965) | $\begin{gathered} \hline \text { Chen } \\ \text { (1975) } \end{gathered}$ | Bolton and | Michalowski (1997) | Frydman and |
|  | (L. B.) | (C) | (U. B.) | Lau | (K) | Burd |
|  | (2) | (3) | (4) | (1993) | (6) | (1997) |
|  |  |  |  | (C) |  | (N. M.) |
|  |  |  |  |  |  | (7) |
|  |  |  |  | (5) |  |  |
| 20 | 1.5599 | 3.16 | 2.7 | 1.60 | 2.332 | NA |
| 25 | 6.5785 | 6.92 | 5.9 | 3.51 | 5.020 | NA |
| 30 | 13.9992 | 15.32 | 12.7 | 7.74 | 10.918 | 8.7 |
| 35 | 31.2645 | 35.18 | 28.6 | 17.80 | 24.749 | 20.7 |
| 40 | 75.7226 | 86.50 | 71.6 | 44.00 | 60.215 | 54.2 |
| 45 | 206.2721 | NA | 195.0 | 120.00 | 164.308 | 153.0 |
| 50 | 663.6980 | NA | NA | 389.00 | 519.089 | NA |

Table 2 Point of Application of $P_{p \gamma}$

| $\phi($ Deg.) | $\theta_{\mathrm{v}}$ (Deg.) | Ratio $=\frac{B-x_{p}}{B}$ |
| :---: | :---: | :---: |
|  |  |  |
| 30 | 4.5243 | 0.1859 |
| 34 | 3.0690 | 0.2140 |
| 36 | 2.4932 | 0.2256 |
| 38 | 2.0038 | 0.2359 |
| 40 | 1.5919 | 0.2451 |
| 42 | 1.2480 | 0.2533 |
| 45 | 0.8442 | 0.2644 |
| 50 | 0.0403 | 0.2799 |

## Conclusions

The following conclusions are drawn from the proposed analysis:

1. An analysis on the evaluation of bearing capacity factor, $N_{\gamma}$ for a smooth footing based on Hill's mechanism is presented.
2. The concept of limit equilibrium condition coupled with Kötter's
equation identifies the unique failure surface consistent with the specified geometry of the failure surface.
3. Application of Kötter's equation makes the analysis statically determinate and facilitates the computation of locating the point of application of passive thrust.
4. The $N_{\gamma}$ values that are obtained from the proposed analysis based on the failure mechanism postulated by Hill with the limit equilibrium approach are the unique values since no other simplifying assumptions are made to evaluate them.
5. The $N_{\gamma}$ values as obtained from the proposed analysis which effectively uses force and moment equilibrium equations show a good agreement with those reported by Sokolovski and Chen; and this establishes reliability of the proposed method of analysis.
6. The point of application of passive thrust $P_{p \gamma}$ can only be computed using the proposed method. It lies closer to the apex of the triangular wedge below the footing. Analysis also shows that the pole of the logspiral lies at footing edge for practical range of $\phi$ values.

## Appendix

Integration for $R_{V}$ and $R_{H}$

$$
\begin{aligned}
& I_{1}=\gamma r_{0}^{2} K \sin (\pi / 4+\phi / 2) e^{3 \theta_{m} \tan \phi}\left[\begin{array}{l}
e^{-\theta_{m} \tan \phi} \sin (\pi / 4-\phi / 2) \\
-\sin \left(\pi / 4-\theta_{m}-\phi / 2\right)
\end{array}\right] \\
& I_{2}=\frac{\gamma r_{0}^{2} \sec \phi}{4\left(1+9 \tan ^{2} \phi\right)}\left[\begin{array}{l}
3 \tan \phi\left\{\sin 2 \phi e^{2 \theta_{m} \tan \phi}+\sin 2\left(\theta_{m}-\phi\right)\right\} \\
\left(e^{2 \theta_{m} \tan \phi}-1\right) \\
\sin \phi
\end{array}\left(\begin{array}{l}
\left.e^{2 \theta_{m} \tan \phi} \cos 2 \phi-\cos 2\left(\phi-\theta_{m}\right)\right)
\end{array}\right]\right. \\
& I_{3}=\frac{\gamma r_{0}^{2} \sec \phi e^{3 \theta_{m} \tan \phi}}{\left(1+9 \tan ^{2} \phi\right)}\left\{e^{-\theta_{m} \tan \phi} \sin (\pi / 4-\phi / 2)-\sin \left(\pi / 4-\theta_{m}-\phi / 2\right)\right\} \\
& I_{4}=\gamma r_{0}^{2} K \sin (\pi / 4+\phi / 2) e^{3 \theta_{m} \tan \phi}\left\{\begin{array}{l}
\cos \left(\pi / 4-\theta_{m}-\phi / 2\right) \\
-e^{-\theta_{m} \tan \phi} \cos (\pi / 4-\phi / 2)
\end{array}\right\} \\
& I_{5}=\frac{\gamma r_{0}^{2} \sec \phi}{4\left(1+9 \tan ^{2} \phi\right)}\left[\begin{array}{l}
\sec \phi\left(e^{2 \theta_{m} \tan \phi}-1\right) \\
-\tan \phi\left(e^{2 \theta_{m} \tan \phi} \cos 2 \phi-\cos 2\left(\phi-\theta_{m}\right)\right) \\
-\left\{\sin 2 \phi e^{2 \theta_{m} \tan \phi}+\sin 2\left(\theta_{m}-\phi\right)\right\}
\end{array}\right]
\end{aligned}
$$

$$
\begin{aligned}
& I_{6}=\frac{\gamma r_{0}^{2} \sec \phi e^{3 \theta_{m} \tan \phi}}{\left(1+9 \tan ^{2} \phi\right)}\left\{\begin{array}{l}
\cos \left(\pi / 4-\theta_{m}-\phi / 2\right) \\
-e^{-\theta_{m} \tan \phi} \cos (\pi / 4-\phi / 2)
\end{array}\right\} \\
&\{3 \tan \phi \sin (\pi / 4+\phi / 2)-\cos (\pi / 4+\phi / 2)\}
\end{aligned}
$$

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[^0]:    * Professor, Dept. of Civil Engineering, I.I.T. Bombay, Mumbai - 400 076. INDIA. Email: dmde@civil.iitb.ac.in
    ** Formerly Research Scholar, Dept. of Civil Engineering, I.I.T. Bombay, Mumbai 400 736. INDIA.
    *** Assistant Professor, Applied Mechanics Department, SVNIT, Surat - 395007. INDIA. Email: vasawant@yahoo.com
    **** Assistant Professor, Civil Engineering Department, D.M. College of Engg., New Mumbai - 400708 INDIA, E-mail:hschore@rediffmail.com.

